Dynamics of the Primary Cilium in Shear Flow

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Abstract

In this work the equilibrium shape and dynamics of a primary cilium under flow are investigated using both theoretical modeling and experiment. The cilium is modeled as an elastic beam that may undergo large deflection due to the hydrodynamic load. Equilibrium results show that the anchoring effects of the basal body on the cilium axoneme behave as a nonlinear rotational spring. Details of the rotational spring are elucidated by coupling the elastic beam with an elastic shell. We further study the dynamics of cilium under shear flow with the cilium base angle determined from the nonlinear rotational spring, and obtain good agreement in cilium bending and relaxing dynamics when comparing between modeling and experimental results. These results potentially shed light on the physics underlying the mechanosensitive ion channel transport through the ciliary membrane.

Key words: Primary Cilium; Stokes Flow; Modeling; Slender Body; Mechanosensing
Introduction

The primary cilium is an isolated non-motile structure found ubiquitously in many non-mitotic mammalian cells (1, 2), depicted schematically in figure 1. As a hair-like protrusion from the apical cell membrane into the extra-cellular space, the cilium axoneme (region I in the figure) consists of nine microtubule doublets that originate from (and are supported by) the basal body. The nine doublets distribute in a radially symmetric fashion, and are enclosed by the ciliary membrane (see figure 1.)

The basal body consists of the mother centriole (with distal and subdistal appendages) and the daughter centriole (see (3) for details). In fact, the basal body is a modified form of the microtubule organizing center of mitotic spindles. When not involved in mitosis, the mother centriole migrates to the cell membrane, where it is attached to the cell membrane through distal appendages and acts as a template of ciliogenesis and an anchor basal body for the primary cilium. The details of the anchorage are incompletely understood but is known to involve \(\gamma\)-tubulin associated with the basal feet, and unique filamentous structures known as striated rootlets (4).

Between the axoneme and the basal body is a transition zone (TZ), labeled as region II in figure 1, where the transition zone fibers (also called ciliary necklace or Y-connectors) bridge the TZ microtubules with the ciliary membrane. These connectors distribute around the TZ microtubules in a more or less symmetrical fashion. As shown in the figure 1, the TZ membrane forms a wedge that extends to the ciliary pocket. Adjacent to the ciliary pocket, the membrane is bound to the distal appendages (5, 6). The TZ is associated with various proteins and molecular motor transport. It is also found that the lipid composition and the permeability of the TZ membrane are slightly different than that of the ciliary membrane (5, 6).

Primary cilia differ from the better understood motile cilia in several important aspects. For example, primary cilia lack (a) a central pair of microtubule doublet, (b) connections between the outer doublets, and (c) other molecular machinery associated with motility. Unlike motile cilia, there is only one primary cilium per cell. Motile cilia are expressed on specialized cells while primary cilia are found on virtually every cell type. Furthermore, primary cilia basal bodies have multiple basal feet and striated rootlets whereas motile cilia have only one of each (7).

Since the initial morphological descriptions of primary cilia over a century ago, their function has only recently begun to be unraveled as a center of chemical and mechanical signal transduction during vertebrate development in bone, kidney, and liver (1). At the beginning the focus has been on
the role of primary cilia in embryonic development where they are known to be involved in establishment of the left-right axis and anterior-posterior limb bud patterning (8–10) via sensing of the hedgehog and wnt families of morphogens.

In the kidney cells, Praetorious and Spring found a dramatic extracellular calcium-dependent increase in intracellular calcium by bending primary cilia of the epithelial cells with fluid flow or micropipette manipulation. They also verified that this response was lost with removal of primary cilia (11, 12). It has been suggested that this response occurs via polycystin-2, a cationic channel that localizes to the base of the cilium (13, 14). This mechanism has also been found in liver cholangiocytes (15). In addition to its role in flow-sensing, the primary cilium is involved in direct transmission of strains in cartilage extracellular matrix. Mechatosensory function of primary cilia has been suggested in other cell types as well, such as human airway smooth muscle and epithelial cells.

To date only a handful of papers have been published on theoretical studies of the mechanical behavior of the primary cilium bending under flow. Schwartz et al. (16) developed a mathematical model based on a small-deformation elastic beam formulation. This model assumed a constant velocity and drag profile along the cilium, which was found to break down under high flow conditions. Resnick et al. (17) applied a similar formulation to study small deflections of the primary cilium in a cylindrical Poiseuille flow. Liu et al. (18) used a more precise model of the fluid flow around an array of cilia by numerically solving Stokes equations. They assumed small rotation at the cilium base even though they compute the drag on cilium axoneme consistently from Stokes equations. Rydholm et al. (19) conducted computational fluid dynamics simulations of the bending of an elastic filament connected to an elastic membrane. From their simulations they found the stress distribution along a filament bent under flow with the maximum stress at the axoneme base. Unfortunately they did not provide any quantitative comparison of cilium bending under flow between simulations and experiments.

In this paper we conduct a quantitative comparison of cilium bending under flow between experiments and modeling. The mechanics of an elastic beam in a viscous fluid flow is investigated using the slender-body theory (SBT) in the limit of small slenderness $\beta \equiv 1/(\ln(\epsilon^2 e)) \ll 1$, where $\epsilon \equiv r/L$ is the ratio of the cross-sectional radius $r$ to the ciliary contour length $L$. In this limit the non-local hydrodynamic interactions are of order $O(\beta)$, and the leading order SBT gives rise to the popular local drag model (20). The local drag model has been widely utilized for studying the dynamics of
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free actin filaments (21, 22) or semiflexible polymers in viscous flow (23). Such leading order SBT is also termed local SBT in (20).

The local drag model is similar to the resistive force theory, which assumes the strength of the singularities is proportional to the local velocity of translation. The main difference lies in the determination of the drag coefficients: In the local SBT the drag coefficients are determined by the leading order terms in the SBT formulation (20, 24), while in the resistive force theory the force coefficients are empirically determined as summarized in (25).

The (local) SBT allows for large bending of the elastic filament under hydrodynamic load (20, 26). In the limit of small filament bending, this model is reduced to those in (16, 17). Following the formulation in Pozrikidis (27, 28), we model the primary cilium as an “anchored” elastic beam by combining SBT and the model for an elastic beam attached to a wall. The boundary condition for the anchored filament end is determined by how the basal body responds to the hydrodynamic load from the fluid flow. For an equilibrium cilium profile under flow, we assume either clamped or hinged boundary conditions for the basal body by specifying the base angle or curvature, respectively.

Using the cilium base angle from the experiment as a boundary condition for the the elastic beam model, we fit the equilibrium cilium profiles to estimate the cilia bending rigidity $E_B$ ( = $EI$, where $E$ is Young’s modulus and $I$ is the moment of inertia of the ciliary axoneme.) These equilibrium results uncover the relationship between the total torque and the cilium base angle as a second-order rotational spring. To elucidate the mechanical origin of such a non-linear rotational spring, we couple the beam-spring system to a cylindrical elastic shell to model the axoneme-membrane system.

The dynamics of the primary cilium toward its equilibrium profile under flow is investigated by integrating our model over time. We further study the tension force distribution along the filament and the elastic shell, and compare it to the critical value for opening a simple mechanosensitive channel, an idealized model for the ion channels that are responsible for the calcium ion transport triggered by bending of primary cilium.

Theory

In the formulation we ignore the inertia effects and consider the Stokes flow regime for the interaction between the primary cilium and the flow. In addition, the aspect ratio of primary cilia $\epsilon$ is often in the range $10^{-2} \leq$
\( \epsilon \leq 10^{-1} \). Consequently the relevant physics is the bending of an anchored elastic slender filament under fluid flow.

In the model we need appropriate boundary conditions for the cilium anchorage that correspond to the mechanical responses of the sub-axonemal compartment (transition zone, basal body, distal and sub-distal appendages, basal feet and rootlets). Due to the complexity of the sub-axonemal compartment, we first model the equilibrium filament attached to a solid wall.

For a given hydrodynamic load, we find that the same equilibrium axonemal profiles can be found with either clamped or hinged boundary conditions at the anchored endpoint.

We adopt the elastic beam formulation and use the hydrodynamic load from the local drag model. The force distribution is denoted as \( \mathbf{F} = F^t(s)\hat{t} + F^n(s)\hat{n} \) with \( s \in [s_0, s_e] \) the arclength, and \( \hat{t} \) and \( \hat{n} \) the unit tangent and normal vectors, respectively. The curvature \( \kappa \) is assumed to be linearly proportional to the moment \( M: M = E_B \kappa \). The external load \( \mathbf{P}(s) = P^t\hat{t} + P^n\hat{n} \) is related to the force by \( \frac{d\mathbf{F}}{ds} + \mathbf{P} = \mathbf{F}_s + \mathbf{P} = 0 \). The moment and the force density is related as \( \frac{dM}{ds} = F^n \). Denoting the filament centerline \( \mathbf{x} = (x(s), y(s)) \) and \( \hat{t} = (t_1(s), t_2(s)) \), the governing equations are

\[
\begin{align*}
x_s &= t_1, \quad y_s = t_2, \quad (1) \\
t_{1s} &= -\kappa t_2, \quad t_{2s} = \kappa t_1, \quad (2) \\
F^t_s &= -\kappa F^n - P^t, \quad (3) \\
F^n_s &= \kappa F^t - P^n, \quad (4) \\
\kappa_s &= -\frac{F^n}{E_B}. \quad (5)
\end{align*}
\]

The local filament inextensibility \( t_1 t_{1s} + t_2 t_{2s} = 0 \) is utilized in deriving equations 2-4. The dimensionless tangential force and the tension force \( \sigma \) are related as \( \sigma = F^t + \kappa^2 \).

We compute the external load \( \mathbf{P} \) from the local SBT as

\[
\mathbf{P} = \frac{\eta (\partial \mathbf{x}/\partial t - \mathbf{U})}{(1 + 2\beta) \mathbf{I} + (1 - 2\beta) \mathbf{x}_s \otimes \mathbf{x}_s}, \quad (6)
\]

where \( \mathbf{U} \) is the fluid velocity at the location \( \mathbf{x} \) in the absence of the elastic filament. \( \eta = \frac{8\pi \mu \dot{\gamma} L^4 \beta}{E_B} \) is the effective viscosity with \( \mu \) the fluid viscosity and \( \dot{\gamma} \) the characteristic flow rate. For the equilibrium filament profile, \( \partial \mathbf{x}/\partial t = 0 \) and equations 1-5 can be solved as a set of boundary-value equations with boundary conditions explained as follows.
At the free filament end \((s = s_e)\), the force-free and torque-free conditions give

\[
F^t(s_e) = 0, \quad F^n(s_e) = 0, \quad \kappa(s_e) = 0. \tag{7}
\]

At the fixed end \((s = s_0)\) \(x(s_0) = 0\) and \(y(s_0) = 0\). If the fixed end is “clamped” the unit tangent vector is specified as

\[
t_1(s_0) = \sin \theta_0, \quad t_2(s_0) = \cos \theta_0, \tag{8}
\]

where \(\theta_0\) is the filament base angle, see figure 2. If the fixed end is hinged with a zero torque, the filament rotates freely and the corresponding boundary conditions are

\[
t_1(s_0)^2 + t_2(s_0)^2 = 1, \quad \kappa(s_0) = 0. \tag{9}
\]

If there is a localized torque at the filament base \((s = s_0)\), the boundary conditions for a hinged filament with a torque at the base are

\[
t_1(s_0)^2 + t_2(s_0)^2 = 1, \quad \kappa(s_0) = \frac{h}{E_B}. \tag{10}
\]

This can be understood from the equation for the moment \(M\) with a localized torque at \(s_0\):

\[
\frac{dM}{ds} = F^n(s) + h \delta(s - s_0), \tag{11}
\]

where \(h\) is the torque magnitude and \(\delta(s - s_0)\) is the Kronecker-delta function at \(s_0\). Integrating over a small interval around \(s_0\), we find that the localized torque induces a jump in the moment

\[
M(s_0^+) = M(s_0^-) + h \rightarrow \kappa(s_0^+) = \kappa(s_0^-) + \frac{h}{E_B} = \frac{h}{E_B}, \tag{12}
\]

which in turn induces a curvature at the cilium base.

For time-dependent filament dynamics, we approximate the time derivative in equation 6 by the second-order time discretization

\[
\frac{\partial \mathbf{x}}{\partial t} = \frac{1}{2\Delta t} \left( 3\mathbf{x}^{k+1} - 4\mathbf{x}^k + \mathbf{x}^{k-1} \right) + O(\Delta t^3) \tag{13}
\]

and the equations for the force distribution at the \(k + 1\) step are

\[
\mathbf{F}^{k+1} + \mathbf{P}^{k+1} = \frac{\eta}{2\Delta t} \left( 3\mathbf{x}^{k+1} - 4\mathbf{x}^k + \mathbf{x}^{k-1} \right) \left( \frac{1}{1 + 2\beta} \mathbf{I} + \frac{1}{1 - 2\beta} \mathbf{x}^k \otimes \mathbf{x}^{k+1} \right), \tag{14}
\]
At every time level we solve the following equations

\begin{align}
    x_{s}^{k+1} &= t_{1}^{k+1}, \quad y_{s}^{k+1} = t_{2}^{k+1}, \\
    t_{1s}^{k+1} &= -\kappa_{s}^{k+1} t_{2}^{k+1}, \quad t_{2s}^{k+1} = \kappa_{s}^{k+1} t_{1}^{k+1}, \\
    \kappa_{s}^{k+1} &= \frac{F_{n,k}^{n,k+1}}{E_{B}}.
\end{align}

Together with the boundary conditions (equations 7 for the free end and one of equations 8-10 for the anchored end).

Two more key ingredients need to be incorporated into the formulation based on the following experimental observations: (I) Before application of flow the unloaded cilia profiles are often curved, which indicates that there is an internal stress in the axoneme. Such internal stress (denoted as \(\sigma_{0}(s)\)) may be due to the structural change of the axoneme or the basal body (31), and can be included in the constitutive relation as \(\kappa_{s} = \frac{F_{n}^{n} + \sigma_{0}(s)}{E_{B}}\), with \(\sigma_{0}(s)\) determined from the profile of the non-stressed cilium. This can be implemented also in equation 17 for the time-dependent model.

(II) The cilium base angle \(\theta_{0}\) varies with time as the cilium bends under flow, as shown in figures 8. This suggests that the fixed angle or curvature assumptions are inappropriate for dynamical modeling. Thus we use a forced over-damped oscillator for the cilium base angle to describe the cilium base rotation

\[ \dot{\theta}_{0} + \gamma \dot{\theta}_{0} + f(\theta_{0}) = \tau, \]

where \(\dot{\theta}_{0} \equiv d\theta_{0}/dt\) is the time derivative of \(\theta_{0}\). \(f(\theta_{0})\) is the restoring torque from the basal anchorage, and \(\tau\) is the torque on the basal body due to the external load on the cilium. At equilibrium \(\dot{\theta}_{0} = \theta_{0} = 0\) and the base angle \(\theta_{0}\) can be determined by finding the roots of equation \(f(\theta_{0}) = \tau = 0\). We determine the functional form of \(f(\theta_{0})\) from the experiment. As shown in Results, the restoring torque \(f(\theta_{0})\) can be fit by a nonlinear quadratic function.

Table I lists the essential parameters in our modeling formulation. Some of the parameters are used later when we couple the elastic filament to a cylindrical elastic shell.

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Description</th>
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</thead>
<tbody>
<tr>
<td>(E_{B})</td>
<td>Bending rigidity of the elastic axoneme</td>
</tr>
<tr>
<td>(\eta)</td>
<td>Effective viscosity (ratio of viscous to elastic forces)</td>
</tr>
<tr>
<td>(E_{T})</td>
<td>Bending rigidity of the cylindrical shell</td>
</tr>
<tr>
<td>(h)</td>
<td>Local torque at the transition region</td>
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</table>
Materials and Experimental Methods

Materials

Inner medullary collecting duct (IMCD) kidney epithelial cells with a stable transfection of GFP somatostatin type 3 fusion protein (SSTR-3) were a gift from Prof. Brad Yoder (University of Alabama at Birmingham) (32). The cells were cultured in DMEM/F12 medium supplemented with 10% fetal bovine serum, 1% penicillin/streptomycin and 200 μg/mL geneticin G4-18 antibiotic (Invitrogen, Carlsbad, CA). Cells were maintained at 37° and 5% CO2. For flow experiments cells were seeded at 75,000 cells/mL on type 1 collagen (33) (BD, Franklin Lakes, NJ) coated 22 × 44 mm No. 1 coverglass (Warner Instruments, Hamden, CT) and grown to 80 – 90% confluency. 48 hours prior to flow and imaging, the cells were serum starved, cultured in DMEM/F12 medium supplemented with 1% penicillin/streptomycin.

Fluid Flow

Fluid shear stress was applied to the IMCD cells in the RC-30 laminar flow chamber (Warner instruments) by a syringe pump (Kent Scientific, Torrington, CT) with a 30 mL syringe (BD). 20 mL of PBS (Invitrogen) was used as the perfusion liquid. The flow chamber had a height $H$ of 250 μm, a width $B$ of 3.2 mm and a length of 38 mm. The flow rates used ranged between 1 and 3 mL/min and were selected to replicate physiological shear stress conditions (34). Flow was instantaneous and unidirectional for 5 seconds and was applied once to each slide. Assuming laminar Stokes flow for a parallel plate flow chamber, the shear stress in the vicinity of the walls for flow between infinitely wide parallel plates depends linearly on the flow rate $\tau = \frac{6 \mu Q}{BH^2}$, where $\tau$ is the shear stress, $\mu$ is the dynamic viscosity and $Q$ is the flow rate (35). The flow velocities near the cilia correspond to a 0.4 – 1.27 Pa wall shear stress. There is some error when calculating the shear forces since the glass slides flex with flow (less than 15 μm in height). Thus our shear stress estimate is only accurate to within ±0.02 Pa.

Imaging

2D $xz$ and 3D $xyz$ image stacks of primary cilia deflecting under fluid flow were captured with a Leica TCS SP5 Laser scanning co-focal. Images were
acquired with a 100×, 1.46 NA oil immersion No. 1 coverglass-aberration-corrected objective, using a 6 kHz resonant bi-directional scanner. The SSTR-3 was expressed along the full length of axoneme, allowing nearly maximum image acquisition of the primary cilium. An Argon laser (with the emitted light peak at 509 nm) was used to excite the GFP fusion protein, and the emitted light was collected at 500 – 600 nm.

Primary cilia were recorded under the fluid flow conditions described above at 10 fps for xz and 4 fps for xyz under 100× optical magnification +10× digital zoom. Cilia were imaged for a total of 2 minutes; 10 seconds before the onset of flow (100 frames for xz, 40 frames for xyz), 5 seconds during the application of flow (50 frames for xz, 20 frames for xyz), and 105 seconds during the recovery period (1050 frames for xz, 420 frames for xyz). For the equilibrium cilium profiles xyz image stacks are taken with a z-step size ranging from 0.3 µm to 0.8 µm: This is because some cilia are shorter than others. For the dynamic data set the xz plane of the cilium was imaged in profile, and the cilium length measurement is limited by (1) out of plane deflection and (2) the determination of the free end point. Both limitations lead to variation of cilium length (less than 0.4 µm) over time.

Before imaging we ensure that no cilia have experienced fluid flow, and only one primary cilium per slide was recorded during flow. 35 slides were utilized in each experimental group and experiments were repeated three times over multiple days, totaling 105 total cilia imaged. Cells were imaged in the center of the flow chamber to reduce wall effects (35).

**Image Processing**

The image stacks were contrast enhanced using Image J software (National Institutes of Health, Bethesda, MD) and processed with Matlab to find the xz (for 2D) xyz (for 3D) coordinates of the primary cilium. Images were converted into 8-bit greyscale image and a Gaussian filter was applied to decrease noise before applying a threshold. The x and y components of the sampled pixels were averaged and recorded as the center of the primary cilium. Primary cilium length was calculated manually by first fitting a polynomial curve through the xz or xyz points and then measuring the length between the endpoints. Noise due to magnification and the cilia fluctuating out of the focal plane caused length variations for individual cilia between images. Not all cilia imaged were used, and cilia which significantly moved out of the focal plane were also excluded in the analysis (34). Rotation at the base of the cilium was calculated by fitting a polynomial curve through
the first four points of the primary cilium image stack and finding the angle between the curve and the normal (glass coverslip). A total number of 80 primary cilia were analyzed.

**Methods of Fitting**

We used the following procedures to systematically find the least-square fits to the experimentally obtained equilibrium cilium profiles.

1. First we normalize the non-stressed cilium profile before the fluid flow is turned on. The contour length is scaled to unity and the internal stress $\sigma_0$ is computed from the non-stressed profile.
2. We then normalize the equilibrium cilium profile under a constant flow, and extract the filament angle at the base, $\theta_0$, from the normalized data points.
3. Both the internal stress $\sigma_0$ and the angle $\theta_0$ are used to fit a clamped cilium to the normalized cilium profile, and a value of $\eta$ is obtained from the least-square fit. Figure 4 show four sets of experimental data superimposed with their corresponding least-square fits.

**Results**

Figures 3 are two $x z$ images of a primary cilium reconstructed from the image stacks. Figure 3(a) is the cilium before the flow is turned on and figure 3(b) is the equilibrium profile of the cilium under flow. The statistics of the bending rigidity $E_B$ for the whole data set of 80 cilia is summarized in figure 5. Almost 70% of the data set falls in the range of $1 \leq E_B \leq 5 \times 10^{-23}$ N-m². The dashed line in the plot indicates the average value, which is higher than reported in (16) and lower than recent results in (31). The average cilium length is 3.9 \(\mu\)m with a standard deviation of 0.8 \(\mu\)m over the 80 primary cilia analyzed.

Combining the cilium base angle (from the experimental data) and total torque (from the least-square fits in our model), we find that the total torque at the base is a non-linear function of the equilibrium cilium base angle displacement. As shown in figure 6, the concave-down dashed line is the least-square quadratic polynomial fit to the full data set (empty symbols). The concave-up solid line is the least-square quadratic polynomial fit to the partial data set (filled symbols). Based on the results in figure 6 we assume that the equilibrium restoring torque from the basal body anchorage takes
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the form

\[ f(\theta_0) = a\theta_0^2 + b\theta_0 + c, \]  \hspace{1cm} (19)

where \( |b| \) is proportional to the linear spring constant, and \( a/b \) is the relative strength of nonlinearity. At equilibrium, the restoring torque balances with the total torque at the cilium base \( f(\theta_{eq}) = \tau \). For the quadratic restoring force in equation 19 there exist two equilibrium angles \( \theta_{eq} \) for a given \( \tau \):

\[ \theta_{eq} = -b \pm \sqrt{b^2 - 4a(c - \tau)} \]

\hspace{1cm} (20)

Linear stability analysis on the equilibrium angle \( \theta_{eq} \) shows that \( \theta_{eq} \) is stable for \( \frac{df}{d\theta}(\theta_{eq}) = f'(\theta_{eq}) > 0 \), and unstable for \( f'(\theta_{eq}) < 0 \). This implies that the equilibrium angle along the concave-up solid curve in figure 6 is always stable. For the concave-down dashed curve, the equilibrium cilium profile with an angle \( \theta_{eq} < 0.3 \) is stable while \( \theta_{eq} > 0.3 \) is unstable. A stable equilibrium angle always exists for \( \theta_{eq} < 0.3 \) for both curves in figure 6. Thus we incorporate the smaller angle in equation 20 into the model. With the coefficients \( (a, b) \) from either of the fits in figure 6, we can determine the cilium base angle for a given torque \( \tau \), either during the bending process or for the equilibrium. An example is illustrated in figure 7.

A non-zero \( c \) in equation 19 means a non-zero restoring torque is needed when the cilium angle is zero. Non-vanishing \( c \) also implies a non-zero base angle when the torque \( \tau = 0 \). The value of \( c \) may be related to the detailed structures of the sub-axonemal compartment, and it may be different for different cells. To elucidate the physiological meaning of this constant, we couple the elastic beam to an elastic cylindrical shell as follows.

The elastic shell surface is parametrized as \((r(l), z(l))\) with \( l \in [l_0, l_e] \) the arc-length along the elastic sheet. \( \psi \) is the angle between the elastic shell normal vector \( \mathbf{n} \) and the \( r \)-axis, see figure 10. The governing equations for the cylindrically symmetric elastic incompressible sheet are (36, 37)

\[ \frac{dr}{dl} = \sin \psi, \quad \frac{dz}{dl} = -\cos \psi, \]

\[ \frac{d\psi}{dl} = -\nu + \frac{\cos \psi}{r}, \quad \frac{d\nu}{dl} = Q, \]  \hspace{1cm} (22)

\[ \frac{dT_m}{dl} = (T_\phi - T_m) \frac{\sin \psi}{r} - \kappa_m Q, \]  \hspace{1cm} (23)

\[ \frac{dQ}{dl} = T_m \kappa_m + T_\phi \kappa_\phi - \frac{\sin \psi}{r} Q, \]  \hspace{1cm} (24)
where $E_T$ is the bending rigidity of the transition membrane. In the following we assume that $E_T \equiv \lambda E_B$ with $\lambda \sim O(1)$. At $l = l_0$ the elastic sheet is connected to the filament, and at $l = l_c$ it is connected to the surrounding membrane.

Equations 21-25 are coupled to equations 1-5 by the boundary conditions at the junction where the filament base $(s = s_0)$ is connected to the shell $(l = l_0)$. Firstly, the unit tangent vector is continuous: The filament tangent vector at the base is related to the angle $\psi$ as

$$t_1(s_0) = -\sin \psi(l_0), \quad t_2(s_0) = \cos \psi(l_0).$$

Secondly, the force distribution and the curvature are also continuous at the junction

$$T_m(l_0) = F_t(s_0) \left( \frac{L_S}{L_B} \right)^2, \quad Q(l_0) = F_n(s_0) \left( \frac{L_S}{L_B} \right)^2,$$

where $L_S$ and $L_B$ are the characteristic lengths for the shell and beam, respectively.

As explained in Introduction, the TZ fibers connect the TZ microtubules to the membrane. Here we assume that the mechanical support from the basal body can be modeled as a localized finite torque (from the transition fibers) on the elastic cylindrical shell at $l = l_0$. In Theory we show that such a localized torque introduces a jump in the moment at the junction: $\mu(l_0) = M(s_0) + h$, and $h/E_B$ is the curvature at the cilium base due to the torque from the TZ fibers.

At the junction the radius of the elastic sheet $r(l_0)$ is taken to be the radius of the cilium, and the height of the transition membrane $z(l_0)$ is assumed to be of the same order as the filament radius. At the opposite end point, the elastic sheet is assumed to be connected to the cell membrane in a flat angle such that $\psi(l_e) = \pi/2$. Equations 21-25 are rendered dimensionless by scaling the length to the filament radius $r_f$ and force to $E_B/r_f^2$.

In our beam-shell model the torque $h$ signifies the mechanical support from the TZ fibers. Figure 11 illustrates the cilium-transition zone profiles for different values of $h$. A non-zero $h$ is needed to support the axoneme in the upright position in the absence of flow because the unforced equilibrium is a flat horizontal shell connected to a horizontal straight beam. As the cilium basal body is mechanically connected to the ciliary membrane through
the connectors and micro-tubular distal appendages, it seems physiologically feasible that non-zero $h$ is related to the support from the transitional fibers that are connected to the ciliary membrane. Numerically we find that the value of $h$ required to support the upright cilium axoneme is proportional to $\lambda$, and in dimensionless units $h = -0.145$ when $\lambda = 1/10$ and $h = -0.402$ when $\lambda = 1/3.6$.

The dash-dotted curve in figure 6 plots the total torque at the base versus the equilibrium angle with $\lambda = 1/3.6$. As discussed in Results, $f'(\theta) > 0$ guarantees a stable equilibrium angle. The concave up curve from the elastic beam-shell model suggests that the data for large positive equilibrium angles may not be reliable. On the other hand, if these large positive angles are reliable, more biological and mechanical features of the sub-axonemal compartment are needed to explain the concave down fit to the nonlinear restoring torque curve (dashed line).

**Dynamics toward equilibrium**

In physiological situations different time scales may be important, such as the time for the cilium to reach the equilibrium profile under flow and the time for the cilium to relax without external load. These time scales are important for the mechano-transduction processes that primary cilia have been implicated in.

As a first step toward understanding how these dynamical processes are related to mechano-sensing, we use our model to simulate the dynamics of cilium bending under flow and cilium relaxation when the flow is turned off. Such dynamical processes can be captured by a high-speed imaging, as shown in figure 8(a) where the flow is turned on at $t = 0$ and the cilium bends as time progresses. In figure 8(b) the flow is turned off at $t \sim 5$ s and the cilium relaxes to the profile labeled at $t = t_f \sim 9.6$ s.

The cilium profile data are obtained by high-speed z-stack scanning of the fluorescence proteins on the ciliary membrane. Consequently noise cannot be estimated and the observed cilium length may not be the same for a single cilium over the duration of 10 seconds (figure 8). In addition, we observe that the initial non-stressed cilium profile in figure 8 is not upright. We also note that after the flow is turned off, the cilium does not relax to the original non-stressed profile, as reported in (31).

In our model simulation, we assume that the rotational spring is over-damped. This implies that at any point in time, the cilium base angle is determined by solving $f(\theta_t) - \tau t_t = 0$. Physically this means that we assume the cilium base angle reaches the semi-equilibrium value “instantaneously”
as the cilium responds to the hydrodynamic environment. This may be an over-simplification worthy of further investigation.

The comparison between experimental and computational findings is summarized in figures 9. The horizontal coordinates of the cilium tip are plotted versus time in figure 9(a). The comparison in cilium profile at different times is shown in figure 9(b). Symbols are the $x$-coordinate of the cilium tip in experiment at different times. To compensate the fluctuating cilium length in the experiments we interpolate or extrapolate the cilium profiles to equal cilium length. The curves are from the simulations. The cilium bends and quickly reaches the maximum bending under flow. The flow is stopped at $t \sim 5$ s and the cilium relaxes to a profile close to the initial one. Despite the noises, results indicate that cilium bending and relaxation is well approximated by our simple model incorporating viscous stress, rotational stiffness and residual stress.

The relaxation time of the cilium is considerably longer than the elastic relaxation time of a free semi-flexible elastic filament (23). This may be understood by taking into account the rotational relaxation time due to the basal body anchorage (such as the shear elastic modulus of the transition membrane due to its coupling with the sub-axonemal compartment). However, more structural details of the ciliary basal body is needed to fully understand the origin of the rotational relaxation at the base.

**Discussion and Conclusion**

The bending of the primary cilium due to fluid flow is known to open membrane ion channels. The molecular details of the transmembrane proteins reveal some aspects of their function and behavior. However, little is known about the mechanical coupling between the bending of a primary cilium under flow and the activation of ion channels.

Quantitative description of simple mechanosensitive channels (MscL) in idealized configurations reveals that a critical tension in the lipid bi-layer membrane is required to activate an ion channel. By incorporating an ion-channel protein into the free-energy formulation of a planar lipid bi-layer membrane, the critical line tension required to keep the protein open can be expressed in terms of the bending rigidity for an individual leaflet, the width of the hydrophobic patch of channel protein, the average size of the ion channel in the open and closed states, and spring constant for the mismatch between the membrane and protein hydrophobic patch. Once the force in the membrane is large enough to overcome the critical force, MscL will be
activated to facilitate the transmembrane transport of ions such as calcium (38–40).

In our calculations we find that the tension force along the elastic filament and the membrane has a maximum at the junction. The tension force along the cilium is plotted in figure 12(a). The corresponding tension force along the transition membrane is shown in figure 12(b). The maximum tension on the transition membrane is much smaller than that along the cilium axoneme. This implies that channels on the axoneme near the base are more likely to be open due to the bending under flow.

To compare this tension with the critical tension required to open MscL, we need to know how the axoneme tension is related to the tension along the ciliary membrane. We also note that the critical force for opening MscL is computed based on several assumptions (38–40). The first key assumption is that the reference lipid bi-layer membrane profile is assumed to be flat. The second assumption is that mechanical force alone is sufficient to open the transmembrane channel. Finally, the cylindrical symmetry assumed for the transition membrane needs to be relaxed for us to make refined comparison.

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References


Dynamics of Primary Cilium in Shear Flow


Figure Legends

Figure 1.
Sketch of the primary cilium structures: Region I is the elastic filament, including both the axoneme and the ciliary membrane. Region II is the transition zone where transition zone fibers connect the transition zone microtubules with the ciliary membrane. Cylindrical geometry is assumed for the transition zone. The cross section of a primary cilium shows the nine pairs of microtubule bundles in the axoneme.

Figure 2.
Cilium axoneme coupled to a rotational spring at the cilium base. Under flow the cilium bends, and the basal body support for the basal body is modeled as a rotational spring described by equation 18.

Figure 3.
Examples of cilium profiles reconstructed from the \(xz\) stacks. Panel (a): Primary cilium profile before the flow is applied. Panel (b): Equilibrium cilium profile under flow.

Figure 4.
Comparison of filament deflection under shear flow between simulation (curves) and experiment (symbols). The cilium is clamped with a base angle \(\theta_0\). \(\eta\) is found from the least-square fit. From top to bottom in the legend, the corresponding cilium lengths are \(2.1 \pm 0.3, 4.7 \pm 0.5, 3.6 \pm 0.4\) and \(2.9 \pm 0.4 \mu m\), respectively.

Figure 5.
Probability distribution of the estimated axoneme rigidity \(E_B\) for the primary cilium equilibrium profiles from the experiments.

Figure 6.
Total torque at cilium base versus the equilibrium cilium angle. \(\Delta \theta \equiv \theta_{eq} - \theta_0(t = 0)\) is the equilibrium angle relative to the initial value. The dashed line is the least-square fit to the full data set (empty symbols), and
the solid line is the fit to the partial data set (solid symbols). The dash-dotted line is from the elastic beam-shell model in Results.

**Figure 7.**

Fitting the equilibrium cilium profile under flow with the non-stressed cilium profile and nonlinear rotational stiffness. The cilium length is $4.1 \pm 0.3 \, \mu m$.

**Figure 8.**

Panel (a): The flow is turned on at $t = 0$. As time progresses to $t \sim 4.6$ s the cilium bends until it reaches a maximum. Panel (b): The flow is turned off and the cilium relaxes from $t = 5$ s until reaching a stress-free configuration at $t = t_f \sim 9.6$ s. The cilium length is $2.7 \pm 0.5 \, \mu m$ at $t = 0$.

**Figure 9.**

Comparison between model and experiment results shown in figure 8. Panel (a) plots the $x$-coordinate of cilium tip versus time, and panel (b) compares the cilium profiles at different times. The cilium length is $2.7 \pm 0.5 \, \mu m$ at $t = 0$.

**Figure 10.**

Coordinate system for the cylindrical shell in the transition zone. Region I is the elastic slender beam, and region II is the transition zone that is bound to the distal appendages (small vertical bars on the right end, where $\psi = \pi/2$ is fixed).

**Figure 11.**

Cilium-transition zone profile in a quiescent flow ($\eta = 0$) with different torque $h$ at the base.

**Figure 12.**

The spatial distribution of the tension force along the cilium with parameters from the fits to the four data sets in figures 4. Panel (b): The spatial distribution of the tension force along the membrane with parameters from the fits to the four data sets in figures 4.
Figure 1:

Cross section of a primary cilium

Region I: elastic cilium

Region II: transition zone

Basal body

Ciliary pocket

Ciliary membrane

A tubule

B tubule

Membrane bound to cytoskeletal network

Plasma membrane
Figure 2:
Figure 3:
Figure 4:
Figure 5:

Average $E_B = 8.4 \times 10^{-23} \text{ Nm}^2$
Figure 6:
Figure 7:
Figure 8: Flow is turned off and the cilium relaxes.
Figure 9:
I: Slender elastic beam

II: Cylindrical elastic shell

Figure 10:
Figure 11:
Dynamics of Primary Cilium in Shear Flow

Figure 12: